18.11 Determine the angular momentum H₀ of the disk of sample problem 18.2 from the expression obtained for its linear momentum m\(\vec{v}\) and its angular momentum H₀, using Eqs. (18.11). Verify that the result obtained is the same as that obtained by direct computation.

omputation.

$$\Sigma_{G} = LT \quad \Sigma_{G/G} = RJ$$

$$\omega_{1} = \omega_{1}T \quad \omega_{2} = -\omega_{2}T$$

$$\omega_{2} = LT - RJ$$

$$\Sigma_{G} = \Sigma_{G} + \Sigma_{G/G} = LT - RJ$$

$$\Sigma_{G} = \Sigma_{G} + \Sigma_{G/G} = (-\omega_{1}R + \omega_{2}L)\hat{E} = 0$$

$$\omega_{1} = \omega_{1}\frac{P}{L}$$

$$\omega_{2} = \omega_{1}\frac{P}{L}$$

$$\begin{split} & \mathcal{I}_{X} = \frac{1}{2} \, m R^{2} - \mathcal{I}_{X} \\ & \mathcal{I}_{y} = \frac{1}{2} \, m R^{2}; \quad \mathcal{I}_{y} = \mathcal{I}_{y} + m L^{2} = \frac{m R^{2}}{4} + L^{2} m \qquad \qquad = \omega_{i} \mathcal{I}_{-} \omega_{2} \mathcal{J} \\ & \mathcal{I}_{z} = \frac{1}{2} \, m R^{2}; \quad \mathcal{I}_{z} = \mathcal{I}_{z} + m L^{2} = \frac{m R^{2}}{4} + m L^{2} \\ & \mathcal{I}_{z} = \frac{1}{2} \, m R^{2}; \quad \mathcal{I}_{z} = \mathcal{I}_{z} + m L^{2} = \frac{m R^{2}}{4} + m L^{2} \\ & \mathcal{I}_{0} = \mathcal{I}_{z} \, \omega_{z} \qquad \mathcal{I}_{0} = \mathcal{I}_{z} \, \omega_{z} = \mathcal{I}_{z} \, \omega_{z} = \mathcal{I}_{z} \\ & \mathcal{I}_{0} = \mathcal{I}_{z} \, \omega_{z} \qquad \mathcal{I}_{0} = \mathcal{I}_{z} \, \mathcal{I}_{0} = \mathcal{I}_{z} \, \mathcal{I}_{0} \\ & \mathcal{I}_{0} = \mathcal{I}_{z} \, \mathcal{I}_{0} = \mathcal{I}_{z} \, \mathcal{I}_{0} = \mathcal{I}_{z} \, \mathcal{I}_{z} \\ & \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} \\ & \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} \\ & \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} \\ & \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} \\ & \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} + \mathcal{I}_{z} \, \mathcal{I}_{z} \\ & \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} + \mathcal{I}_{z} \, \mathcal{I}_{z} \\ & \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} + \mathcal{I}_{z} + \mathcal{I}_{z} + \mathcal{I}_{z} \, \mathcal{I}_{z} \\ & \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} + \mathcal{I}_{z} + \mathcal{I}_{z} + \mathcal{I}_{z} + \mathcal{I}_{z} + \mathcal{I}_{z} \\ & \mathcal{I}_{z} = \mathcal{I}_{z} \, \mathcal{I}_{z} + \mathcal{I}_{z} + \mathcal{I}_{z} + \mathcal{I}_{z} + \mathcal{I}_{z} + \mathcal{I}_{z} \\ & \mathcal{I}_{z} + \mathcal{I}_{z} \\ & \mathcal{I}_{z} + \mathcal{I}_{z} \\ & \mathcal{I}_{z} + \mathcal{I}_{z} \\ & \mathcal{I}_{z} + \mathcal{I}_{z} \\ & \mathcal{I}_{z} + \mathcal{I}_{z$$

$$\begin{split} \mathcal{H}_0 &= \mathcal{H}_G + \Sigma_C \times (m \, \mathcal{V}_G) \\ \mathcal{H}_G &= \Xi_X \, \omega_X \, \mathcal{I} + \Xi_3 \, \omega_y \, \mathcal{I} + \Xi_2 \, \omega_2 \, k = \frac{1}{2} \, m \, R^2 \omega_1 \, \mathcal{I} - \frac{1}{4} \, m \, R^2 \omega_1 \, \frac{R}{L} \, \mathcal{I} \\ \Sigma_G \times m \, \mathcal{K}_G &= L \, \mathcal{I} \times (m \, R \, \omega_1 \, \mathcal{L}) = -m \, R \, L \, \omega_1 \, \mathcal{J} \\ \mathcal{H}_0 &= \frac{1}{2} \, m \, R^2 \omega_1 \, \mathcal{I} - \left(\frac{1}{4} \, m \, R^2 \, \omega_1 \, \frac{R}{L} + m \, R \, L \, \omega_1 \right) \, \mathcal{J} \quad \text{checks} \end{split}$$

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Introduction to Machine Dynamics Mehrdaad Ghorashi, 2025-09-26 This book introduces machine dynamics an essential competency important for many applications such as designing robots for manufacturing materials handling or the landing gear of an airplane analyzing the motion of a piston in an internal combustion engine or a compressor and designing a Mars Rover Wherever a machine is used for force or power transmission or a mechanism creates a desired motion the methods studied in this book provide the fundamental knowledge needed for optimal design Specific sections are provided on different types of mechanisms and conditions that should be met for obtaining a desired performance including kinematic analysis of mechanisms using direct differentiation relative motion kinematic coefficients and instantaneous centers The Newton Raphson method for solving complex nonlinear position analysis problems is discussed and the determination of dead and limit positions in mechanisms is presented The relation between the angular velocity ratio theorem and the fundamental law of gearing is shown to provide a bridge between the concept of instantaneous centers and analyzing gears Gears and gear trains are covered in detail and calculation of gear ratios in fixed axis and planetary gear trains using the rolling contact equations is illustrated Finally power and force transmission in machines is covered Static and dynamic cases are analyzed and the author shows how the static solutions can provide approximations for the dynamic problems where inertia effects are not significant low inertia and low accelerations Application of matrix algebra for solving the system of equations of equilibrium in statics or equations of motion in dynamics is also illustrated Because of the importance of balancing in any application involving rotating machinery static and dynamic balancing are analyzed. The book concludes with a brief coverage of three dimensional dynamics including Euler's equations and gyroscopic effect Aimed at engineering students interested in machine dynamics across a range of disciplines the book is also ideal as a reference for practicing engineers with a good understanding of statics dynamics and matrix algebra **Earthquake Engineering Research Center Library Printed** Catalog University of California, Berkeley. Earthquake Engineering Research Center. Library, 1975 Decimal Classification and Relativ Index for Libraries, Clippings, Notes, Etc Melvil Dewey, 1919 Library Notes Melvil Dewey, 1898 Abridged Decimal Classification and Relativ Index for Libraries, Clippings, Notes, Etc Melvil Dewey, 1894 Library Notes; Improved Methods and Labor-Savers for Librarians, Readers and Writers ,1895 **European Science Notes** Military Engineering: (1923). Roads Great Britain. War Office,1923 **Notes on Dynamics for Aerodynamicists** ,1993 Julius David Schetzer, 1951 Military Engineering: (1923). Demolitions and mining Great Britain. War Office,1923 Statics with MATLAB® Dan B. Marghitu, Mihai Dupac, Nels H. **Engineering Mechanics I** Y. M. Haddad, 2011 Madsen, 2013-06-13 Engineering mechanics involves the development of mathematical models of the physical world Statics addresses the forces acting on and in mechanical objects and systems Statics with MATLAB develops an understanding of the mechanical behavior of complex engineering structures and components using MATLAB to execute numerical calculations

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